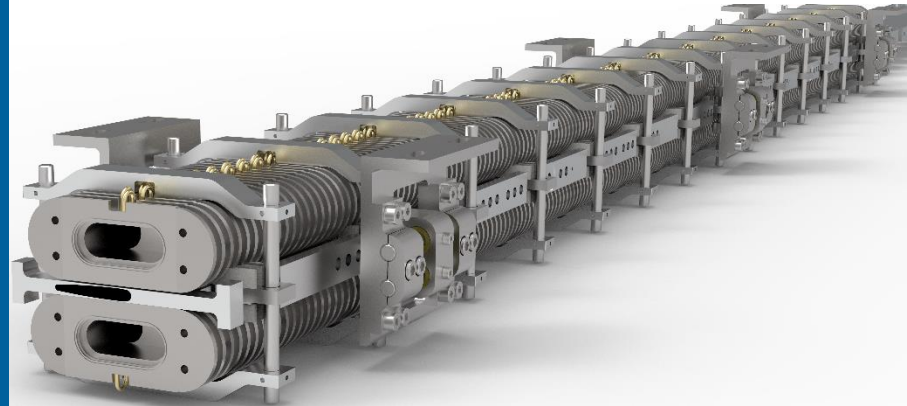


MAGNETIC MEASUREMENT SYSTEM BEING DEVELOPED FOR THE SCU PROGRAM AT THE APS



Matt Kasa and Yury Ivanyushenkov

IMMW 21 Grenoble, France June 24th – 28th

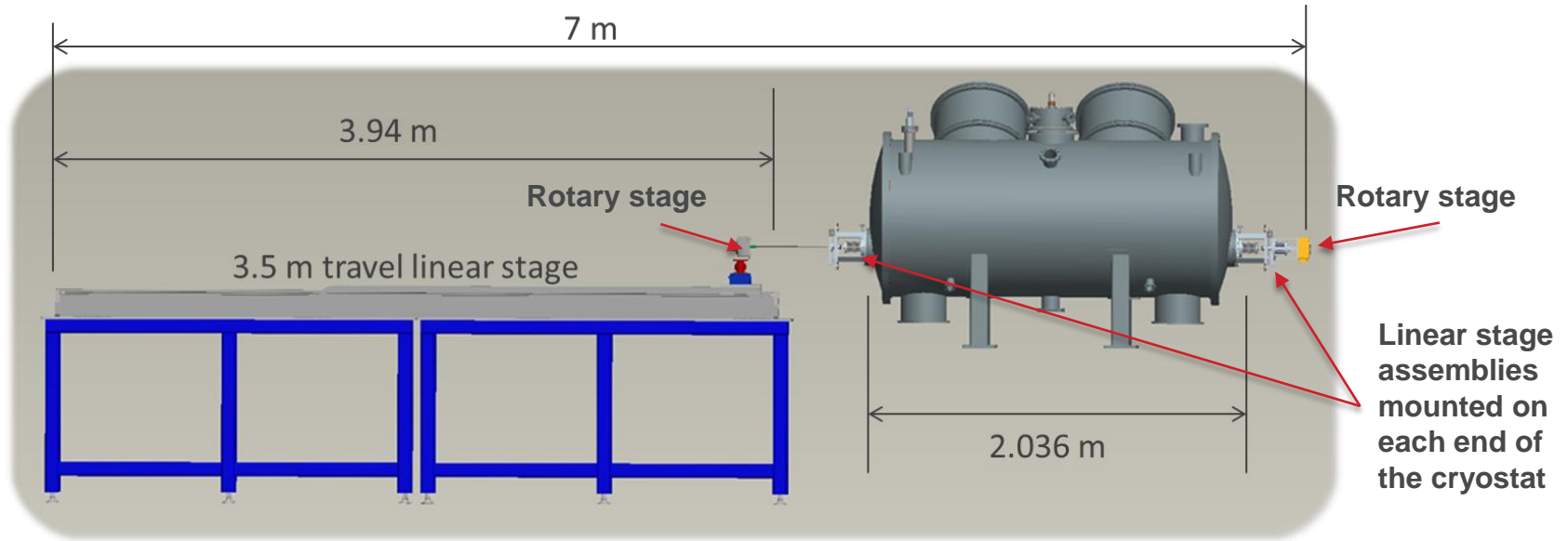
Work supported by the U.S. Department of Energy, Office of Science, under Contract No. DE-AC02-06CH11357.

OUTLINE

- Existing measurement system
- New measurement system guide tube
- New Hall probe drive system
- Wire based measurement system
- Commissioning and initial measurement results

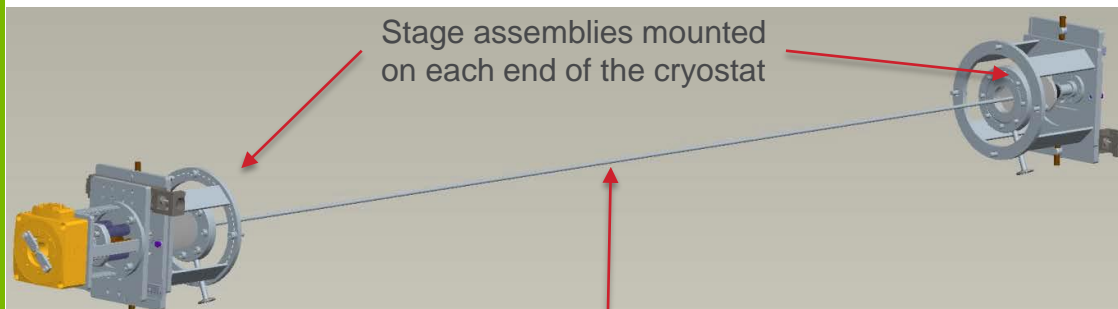
- Thank you
 - Ethan Anliker for design work and providing rendered images
 - Yuko Shiroyanagi for Ansys analysis
 - Isaac Vasserman
 - Roger Dejus

EXISTING SCU MEASUREMENT SYSTEM



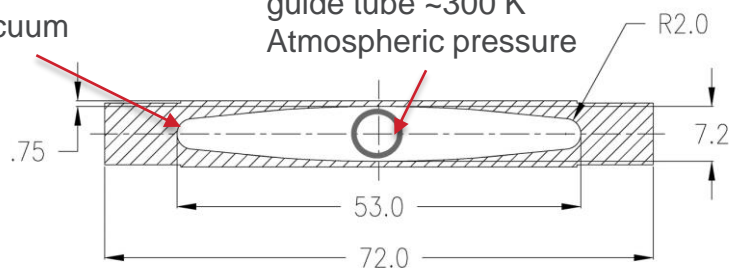
- Adapted from Budker Institute by C. Doose
- Hall probe mounted inside a long carbon fiber tube and scanned through the device using the 3.5 m linear stage
- Wire based measurements performed using rotary stages

EXISTING SCU MEASUREMENT SYSTEM



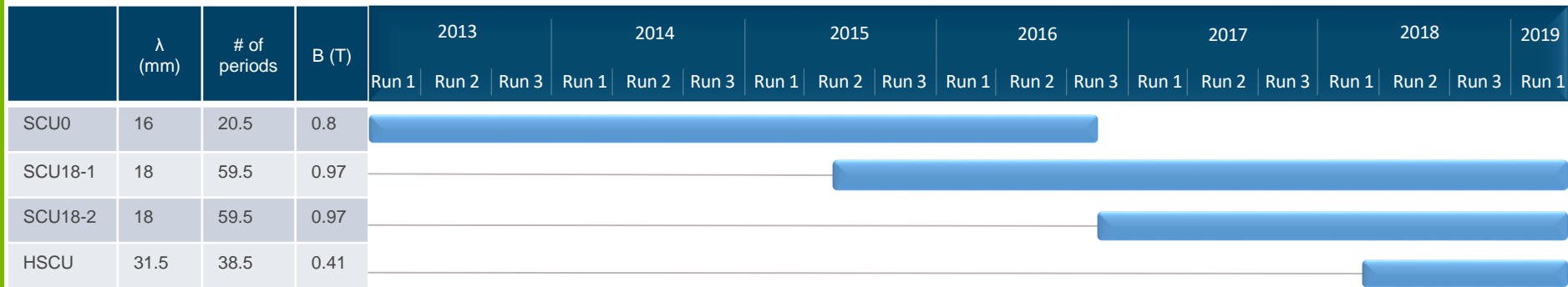
Al beam chamber ~15 K
Under vacuum

6.4 mm OD Titanium
guide tube ~300 K
Atmospheric pressure



- Ti guide tube is isolated from the beam chamber with spiral wrapped Kevlar string
 - Tube is tensioned to reduce sag
 - Heated to room temperature with current
 - Translated horizontally using stages on the cryostat
- Successfully used to measure all of the SCUs currently in operation at the APS
 - Main challenge was to maintain a uniform temperature along the length of the guide tube

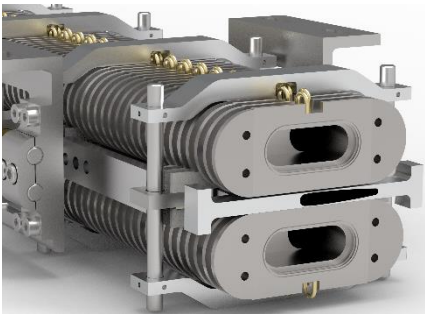
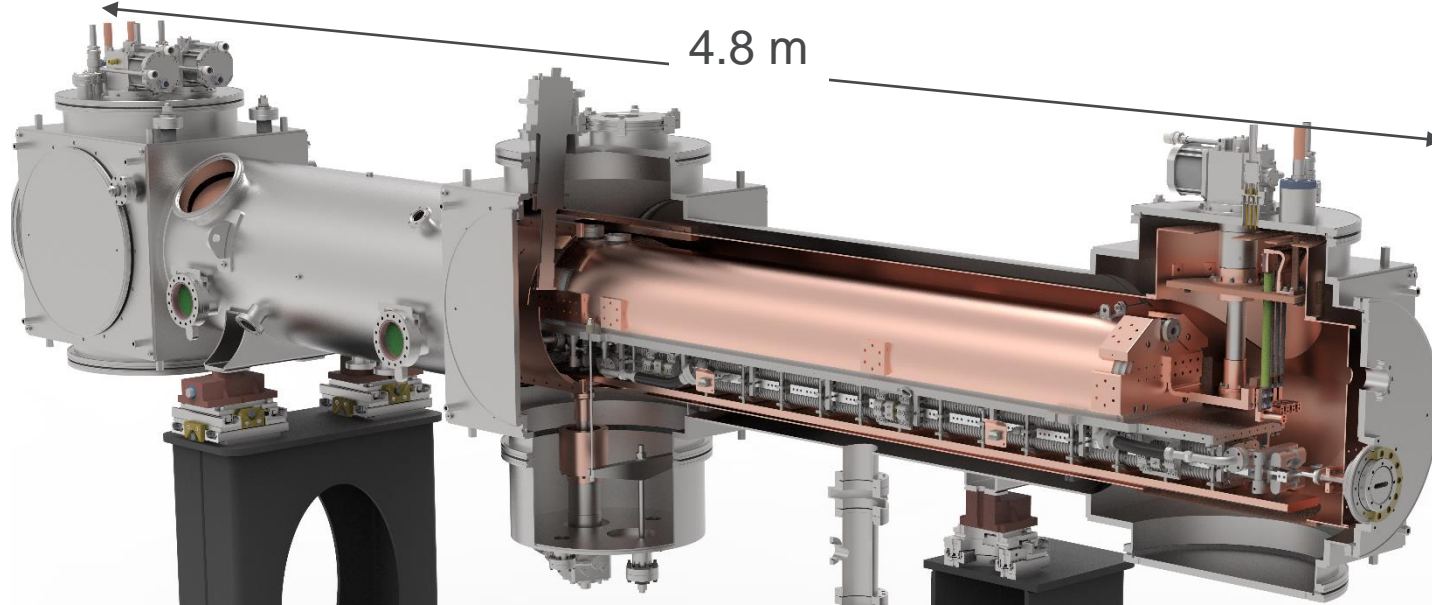
SCUS IN OPERATION AT THE APS



Year	APS delivered	SCU0 and SCU18-2				SCU18-1				HSCU			
		Oper.	Down	quench	avail. %	Oper.	Down	quench	avail. %	Oper.	Down	quench	avail. %
2013	4871 h	4189 h	20 h	34 + 3	99.5	-	-	-	-	-	-	-	-
2014	4926 h	4391 h	174 h [1]	32 + 2	96.2	-	-	-	-	-	-	-	-
2015	4940 h	4834 h	0 h	26 + 1	100	3059 h [2]	0.1 h	5 + 0	99.997	-	-	-	-
2016	4941 h	4647 h [3]	0 h	9 + 0	100	4585 h	0.3 h	11 + 1	99.990	-	-	-	-
2017	4840 h	4756 h	0 h	8 + 1	100	4818 h	0.75 h	13 + 2	99.984	-	-	-	-
2018	4853 h	4755 h	5 h	14 + 1	99.89	4710 h	0.59 h	14 + 2	99.987	751 h	0 h	0 + 0	100
2019 [4]	1691 h	1562 h	4.3 h	7 + 1	99.68	1671 h	0 h	4 + 0	100	144 h	0 h	0 + 0	100
Total	31062 h	29134 h	203.3 h	122 + 8	99.31	18843 h	1.74 h	40 + 5	99.991	895 h	0 h	0 + 0	100

- e-beam has never been lost due to self-quenches
 - **Red = beam dump-induced quench**
 - **Blue = non-beam dump, possible self-induced quench**
- [1] November: Partial loss of one cryocooler capacity
 [2] Installed in May; operated May – Dec. 2015
 [3] SCU18-2 replaced SCU0 in Sep.
 SCU0 3310 h, SCU18-2 1337 h
 [4] January 2019 through April 22, 2019

APSU CRYOSTAT LAYOUT



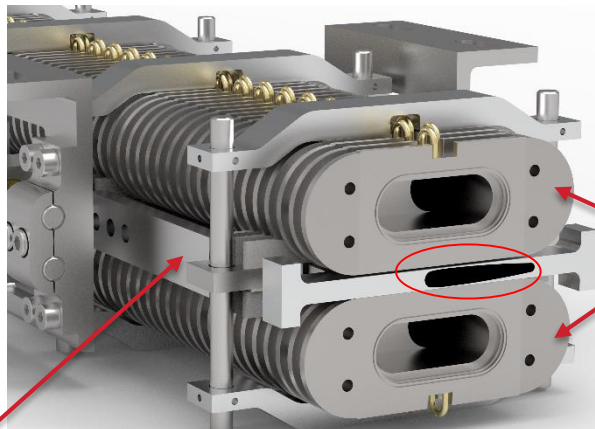
- Four long SCU cryostats
 - 2 in-line and 2 canted
 - 16.5 mm or 18.5 mm period wound with NbTi superconductor
- Two ~1.9 m long magnetic structures
- Magnetic gap is 8 mm
- Vertical beam stay clear is 6.3 mm

SCU MEASUREMENT SYSTEM UPGRADE

- Retain features of the current measurement system
 - Capability to measure the SCUs in the production cryostat under normal operating conditions
 - All measurement components at room temperature and atmospheric pressure

- Issues to address
 - Maintain uniform temperature along the guide tube
 - Affects the straightness of the guide tube
 - Caused the carbon fiber tube of the Hall probe to twist during a scan
 - Upgrade the drive system to accommodate Hall probe scan length ~4.8 m
 - Not easily achieved with the current system and methods
 - Decouple the drive system from the cryostat

EXTRUDED AND MACHINED GUIDE TUBE

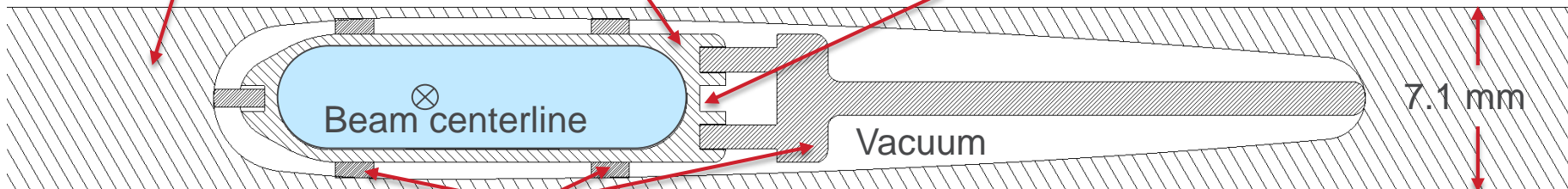


Magnets 4.2 K,
8 mm gap

Aluminum beam
chamber ~20 K

Aluminum guide tube
~300 K, 1 atm

Channel for
heater wires



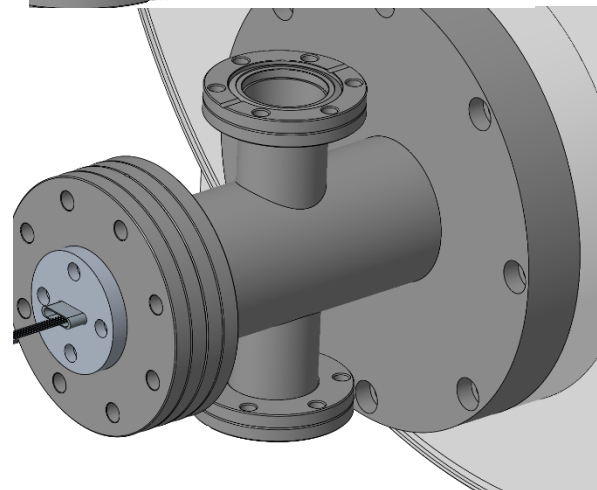
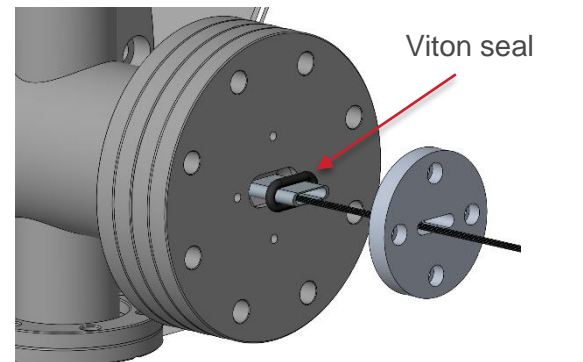
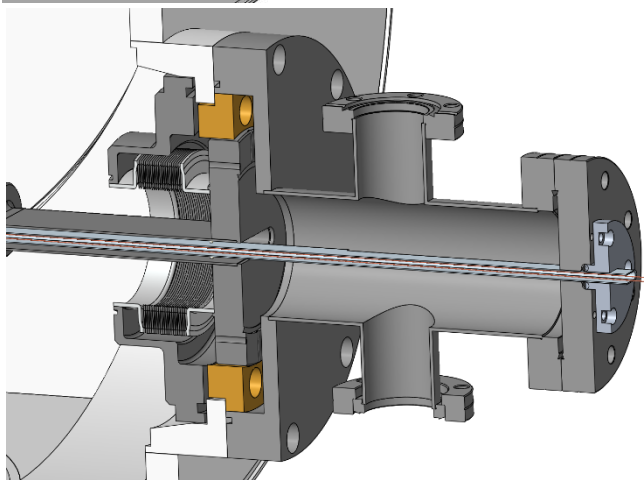
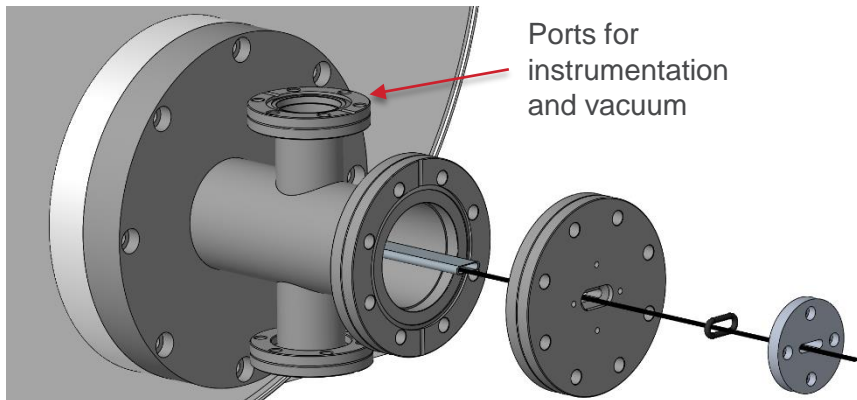
Beam centerline

Vacuum

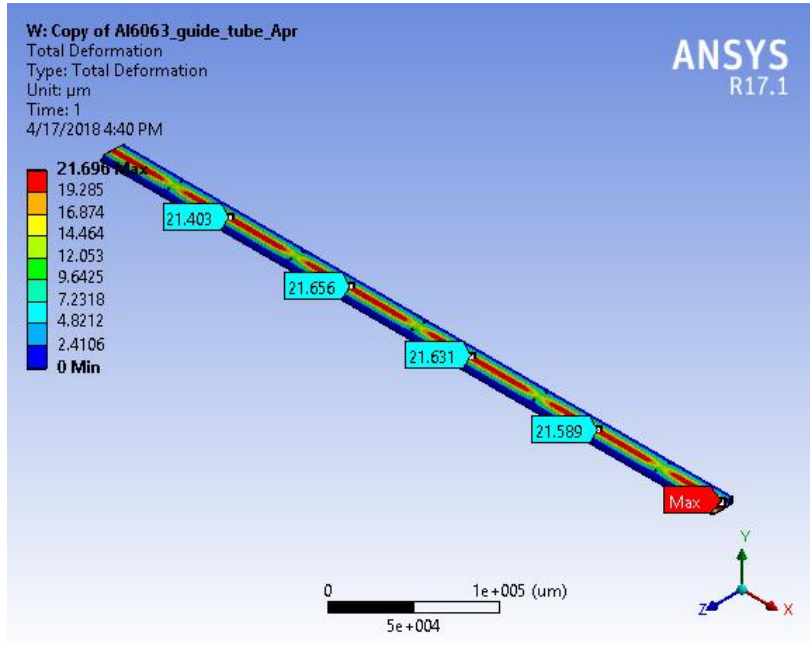
7.1 mm

Torlon standoffs

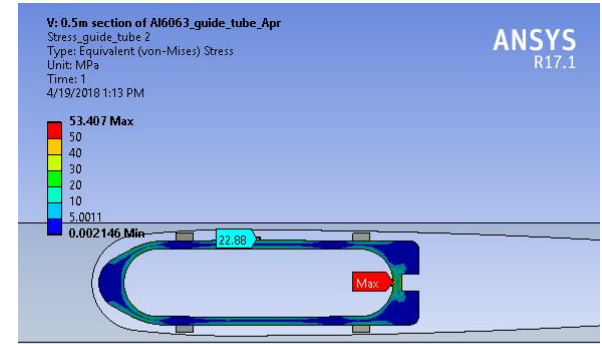
MACHINED GUIDE TUBE



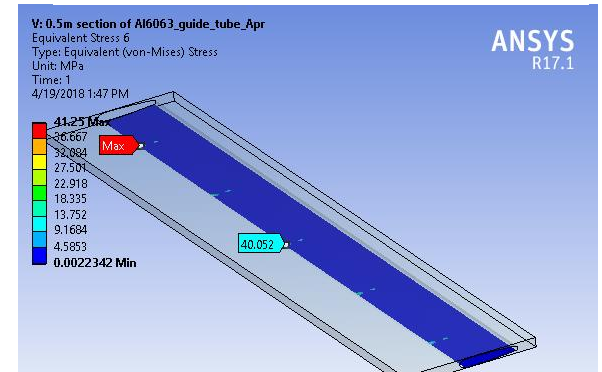
DEFORMATION AND STRESS



Maximum deformation of guide tube = 22 μm
 Beam chamber deformation = 4 μm

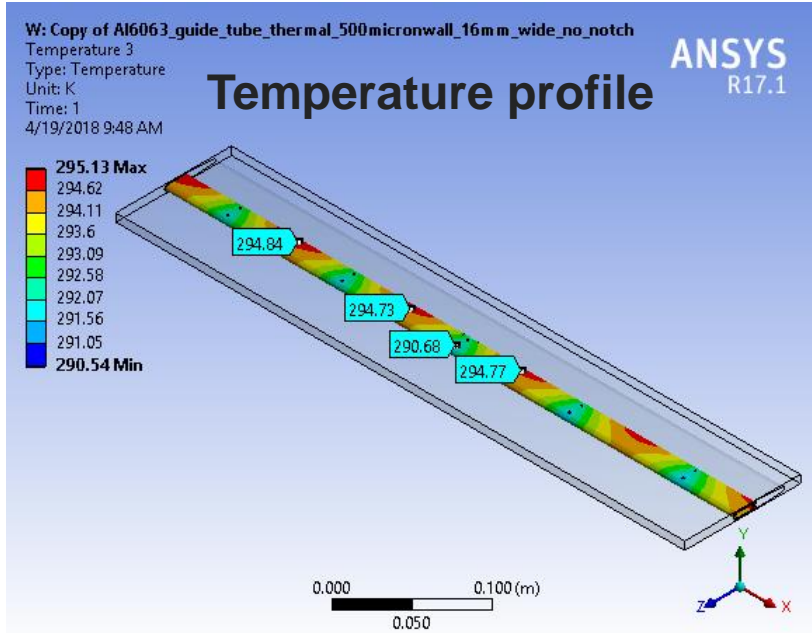


Maximum stress on guide tube = 53 MPa < 145 MPa (yield strength)



Maximum stress on beam chamber = 41 MPa < 145 MPa (yield strength)

HEATER POWER REQUIREMENTS

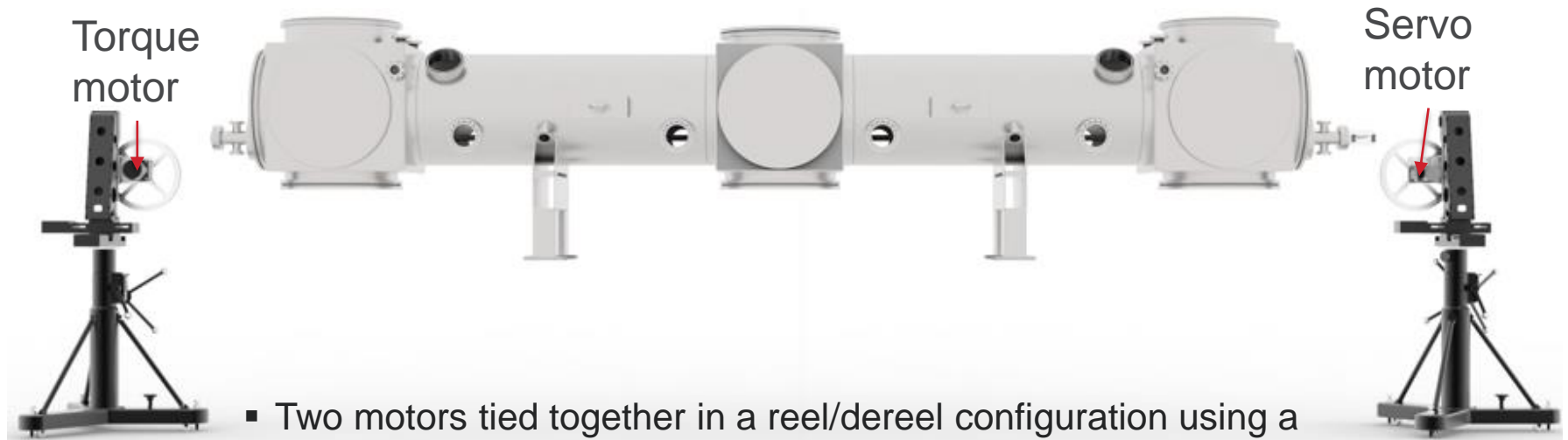


Location	Heat (W)
Q_{total}	2.7
$Q_{\text{conduction}}$	1.78
$Q_{\text{radiation}}$	0.92

- Model: $2.7 \text{ W}/0.5 \text{ m} = 5.4 \text{ W/m}$
- Length of the beam chamber = 4.8 m
- Total heater power = $5.4 \text{ W/m} \times 4.8 \text{ m} = 25.9 \text{ W}$
- 9.6 m long 32 AWG phosphor bronze wire = 38.4Ω
- Power supply requirements: $\sim 0.8 \text{ A}$ at 32.4 V

	AWG	Resistance (Ω/m)			Diameter (mm)	Fuse current air (A)	Fuse current vacuum (A)	Number of leads	Name	Insulated diameter (mm)	Insulation type	Insulation thermal rating (K)	Insulation breakdown voltage (VDC)
		4.2 K	77 K	305 K									
Phosphor bronze	32	3.34	3.45	4.02	0.203	4.2	3.1	1	SL-32	0.241	Polyimide	500	400
								2	DT-32	0.241	Polyimide		
								3	QT-32	0.241	Polyimide		
								4	QL-32	0.241	Polyimide		
	36	8.56	8.83	10.3	0.127	2.6	1.4	1	SL-36	0.152	Formvar®	378	250
								2	DT-36	0.152	Polyimide	500	400
							4	QT-36	0.152	Formvar®	378	250	
								4	QL-36	0.152	Polyimide	500	400

HALL PROBE DRIVE SYSTEM

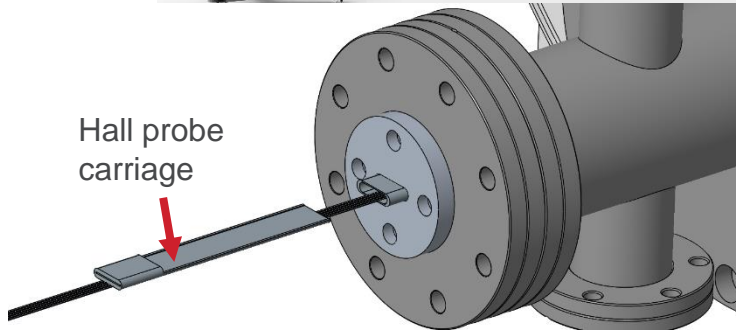


- Two motors tied together in a reel/dereel configuration using a flexible linear encoder scale, Hall probe carriage, and wire
- Hall probe carriage driven through the guide tube
- Servo motor position control through feedback from linear encoder
- Torque motor maintains tension
- System eliminates the need for a long linear stage

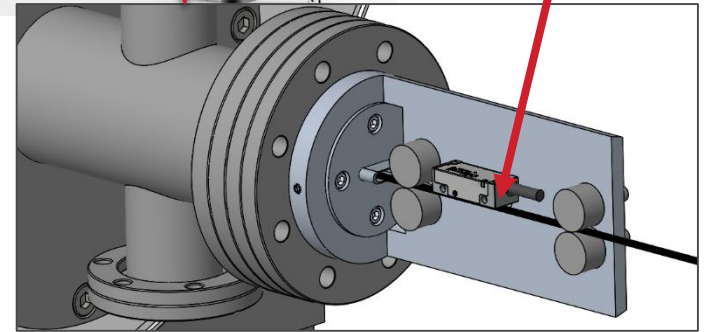
HALL PROBE DRIVE SYSTEM



Encoder read head
and flexible linear
scale



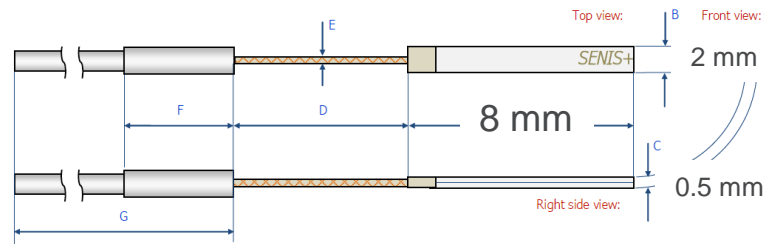
Hall probe
carriage



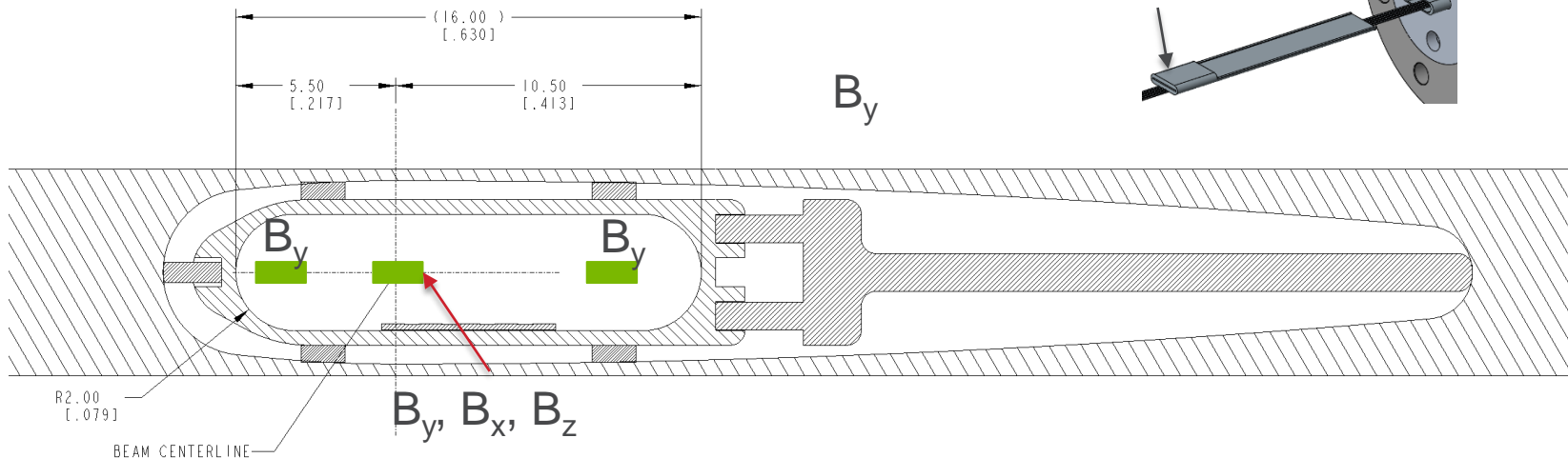
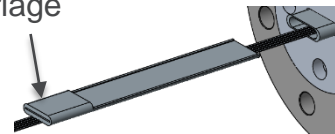
- Veratus series encoder from Celera Motion
 - Flexible scale
 - Inconel 625 – non-magnetic
 - 6 mm x 0.2 mm
 - 1 μm resolution
 - Up to 20 m lengths

SENIS HALL PROBES

- Type H probe, similar to probes we currently use
 - Orthogonality error 3-axis probe <math>< 0.1^\circ</math>
 - Offset and drift error <math>< 1\text{ G}</math>
 - Total measurement accuracy better than 0.1%
 - Integrated temperature sensor
- B_z is used to measure longitudinal field and determine vertical position of sensor in the magnetic gap



Hall probe carriage

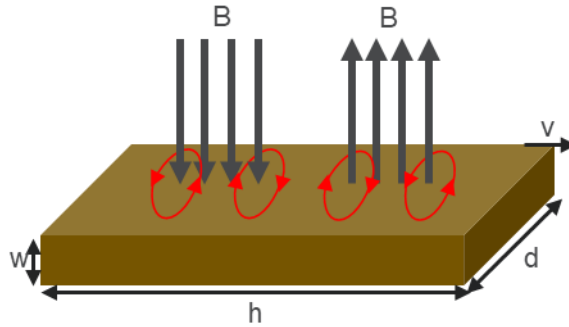


EDDY CURRENT HEATING AND BRAKING FORCE IN THE ENCODER SCALE

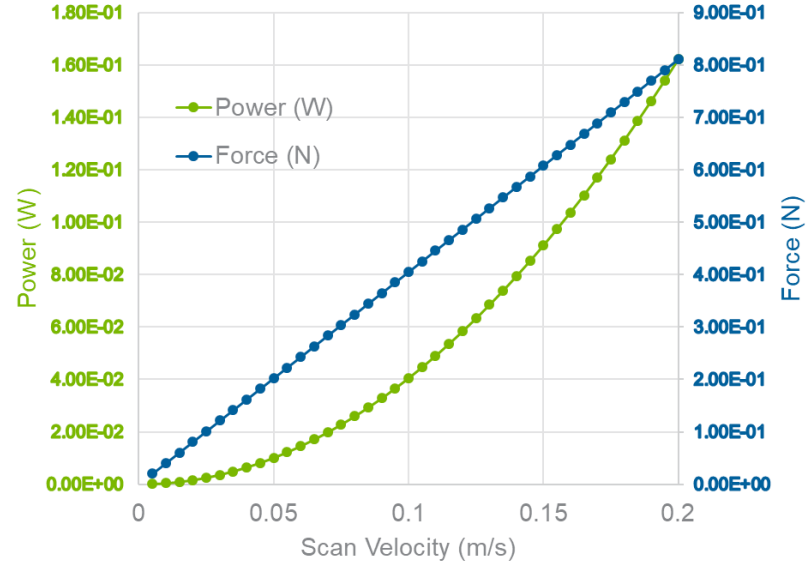
$$P_{EC} = N \frac{v^2 B^2 d h w}{\rho}$$

$$F_{Brake} = N \frac{v B^2 d h w}{\rho}$$

P is power (W)
 B is peak magnetic field (T)
 d is the width of the encoder (m)
 v is the scan velocity (m/s)
 ρ is resistivity of the material (Ω-m)
 w is the thickness of the encoder (m)
 h is the length of one period (m)
 N is the number of periods



- Eddy currents penetrate the full thickness of the scale at all reasonable scan rates.
- Inconel 625 scale properties:
 - D = 8440 kg/m³
 - ρ = 1.29E-6 Ω-m
 - d = 0.006 m
 - w = 0.0002 m
 - μ = 1.26E-6 H/m
 - μ_r = 1.0006

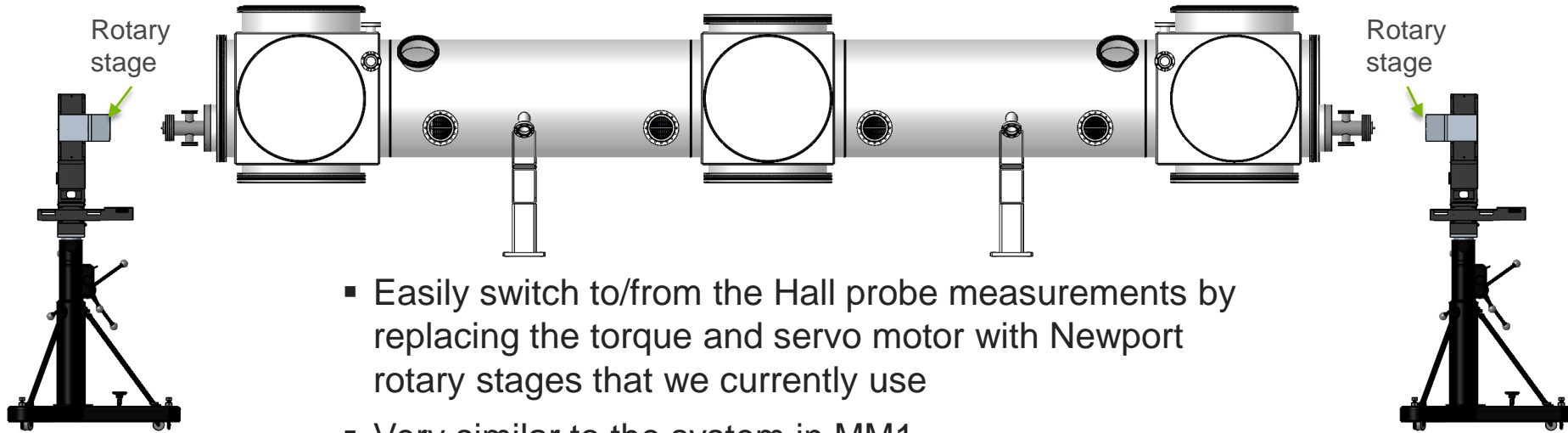


- Two 1.8 m long, 16.5 mm period devices, B = 1.1 T
- Anticipated scan velocity is <0.1 m/s

EFFECTS OF EDDY CURRENT HEATING AND BRAKING FORCE

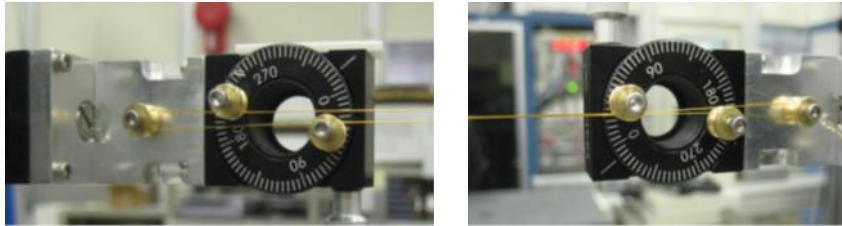
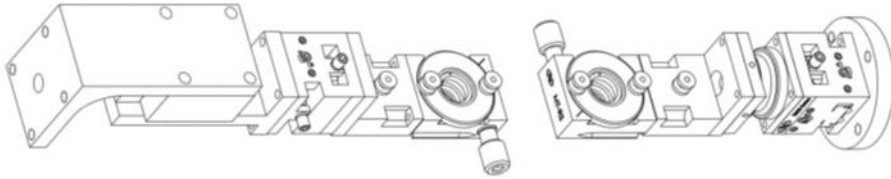
- Temperature rise during a scan due to Eddy current power dissipation
 - Scan 3.6 m long device at 0.1 m/s deposits 1.44 J into the scale
 - Specific heat capacity 0.41 J/g-°C
 - $\Delta T = \frac{1.44}{0.41 \times 36} = 0.098 \text{ }^\circ\text{C}$
- Change in scale length due to temperature rise
 - CTE is 12.8 $\mu\text{m/m-}^\circ\text{C}$
 - $\Delta L = 12.8 \times 3.6 \times 0.098 = 4.4 \mu\text{m}$
- Tension limits
 - Applied braking force is < 1 N
 - Yield strength of Inconel 625 is 460 Mpa
 - 0.006 m x 0.0002 m gives a yield strength of 552 N (124 lbf)
 - Max tension from torque motor is 65 N (14.7 lbf)

WIRE BASED MEASUREMENTS

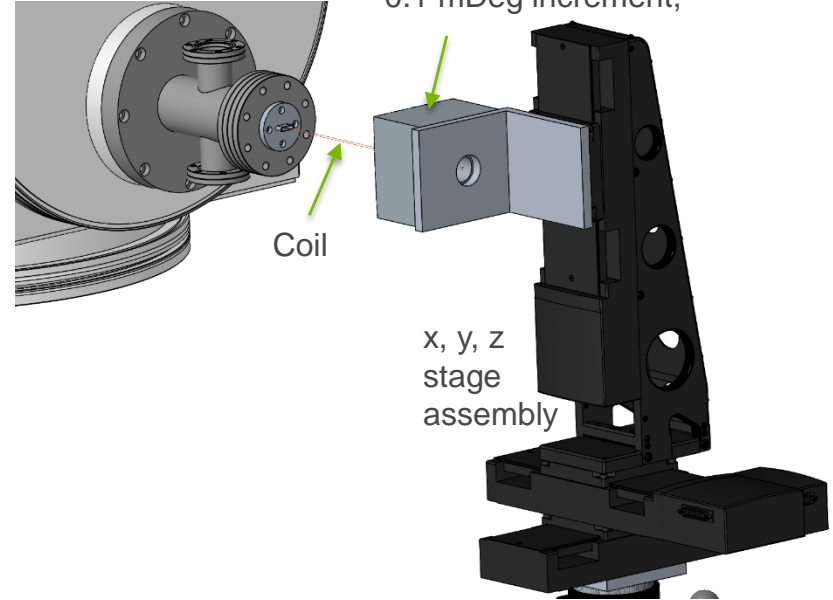


- Easily switch to/from the Hall probe measurements by replacing the torque and servo motor with Newport rotary stages that we currently use
- Very similar to the system in MM1
- Can also be used for pulsed wire measurements by moving one stage assembly further from the cryostat

WIRE BASED MEASUREMENTS



- Implement the same coil holder system developed in MM1
 - Easily adjust the geometry of the coil
- Coil is positioned using the x, y, and z stages
 - Guide tube position remains constant



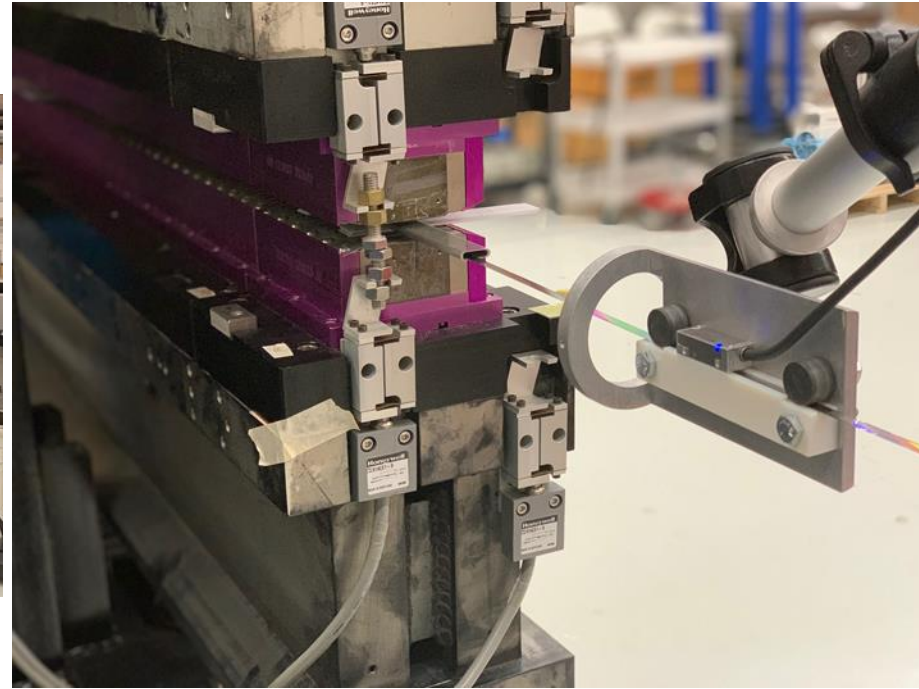
Rotary stage:
RGV100
0.1 mDeg increment,

Coil

x, y, z
stage
assembly

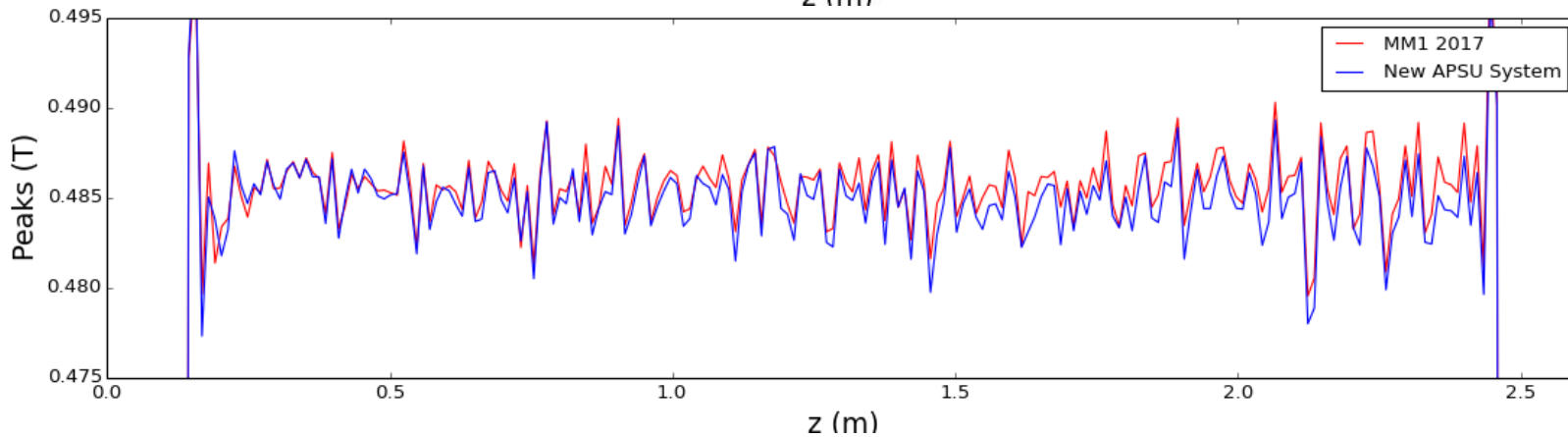
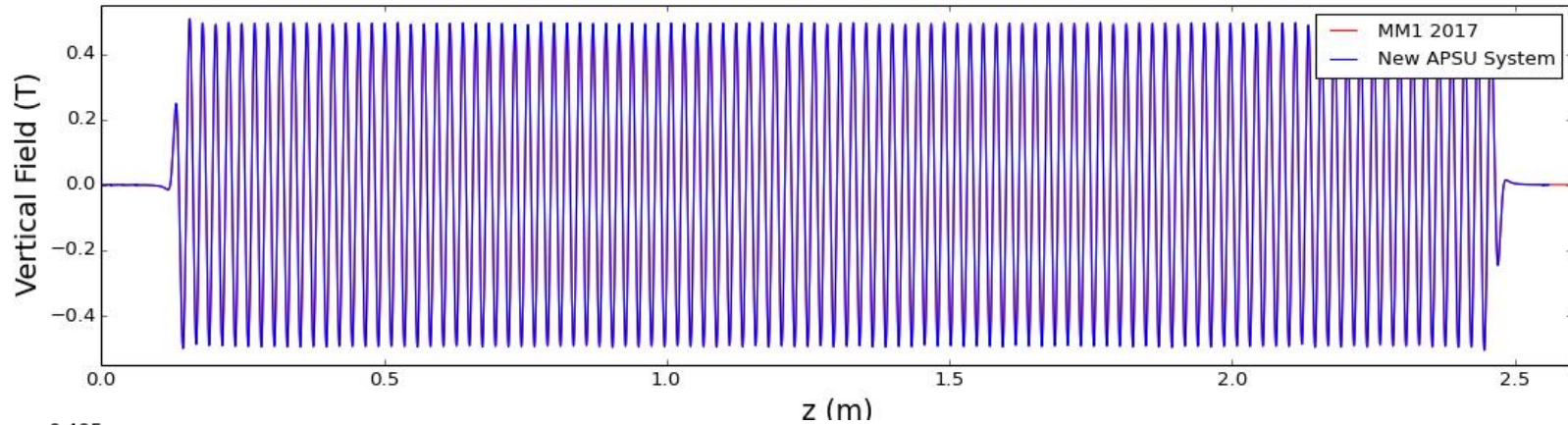
ILS100CC stages:
100 mm travel (± 3 mm spec)
1 μ m resolution

COMMISSIONING



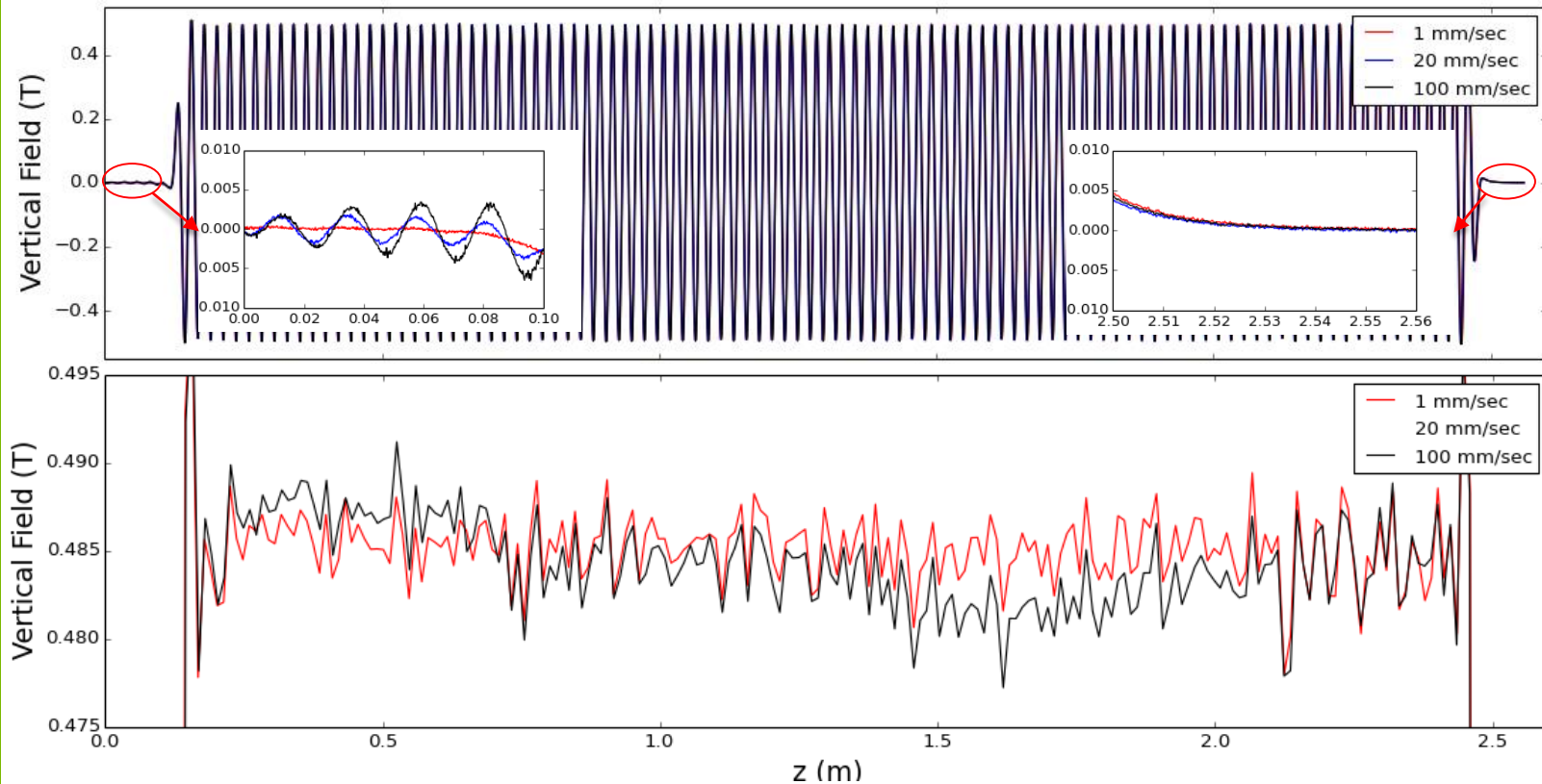
- Measurement system was setup to measure a 2.3 cm period hybrid permanent magnet undulator that was last characterized in MM1 in 2017 by I. Vasserman
- Guide tube was shimmed to be near the center of the gap (11.5 mm)

FIELD SCAN AND FIELD PEAKS



SCANNING SPEED

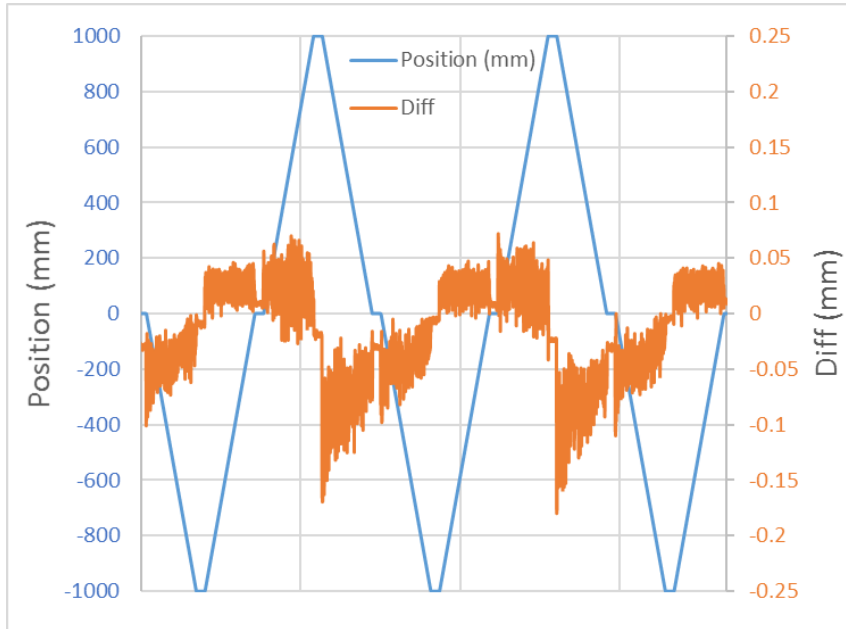
- Data is currently acquired on-the-fly and scanning speeds below 10 mm/sec are required to get reasonable agreement with data from 2017
- Noise increases with scanning speed due to induced voltage on Hall sensor signal cables



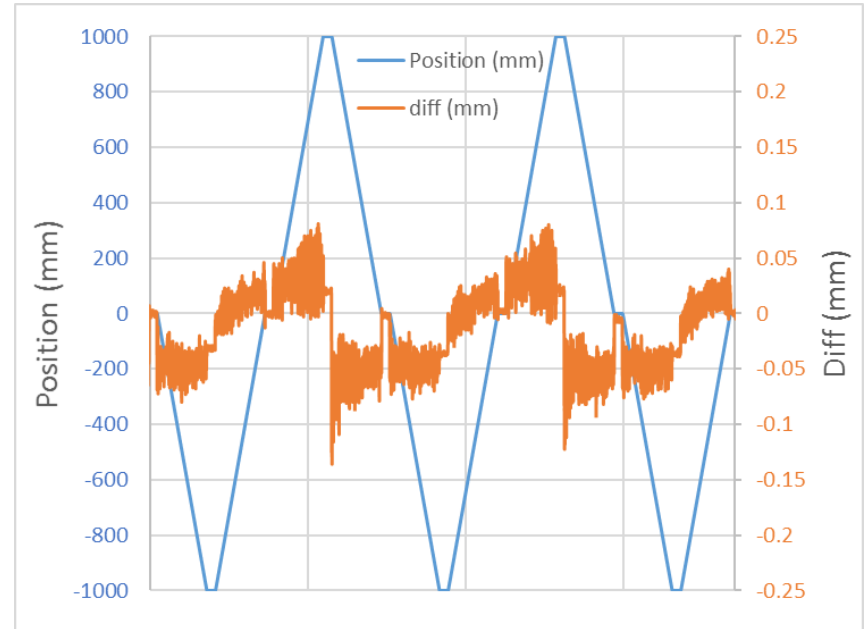
20 mm/sec peaks were suppressed to declutter the plot

TENSION

- Tension affects position accuracy
 - Different tension is needed depending on scan direction
- Plots show the scale position and difference between the scale and interferometer positions

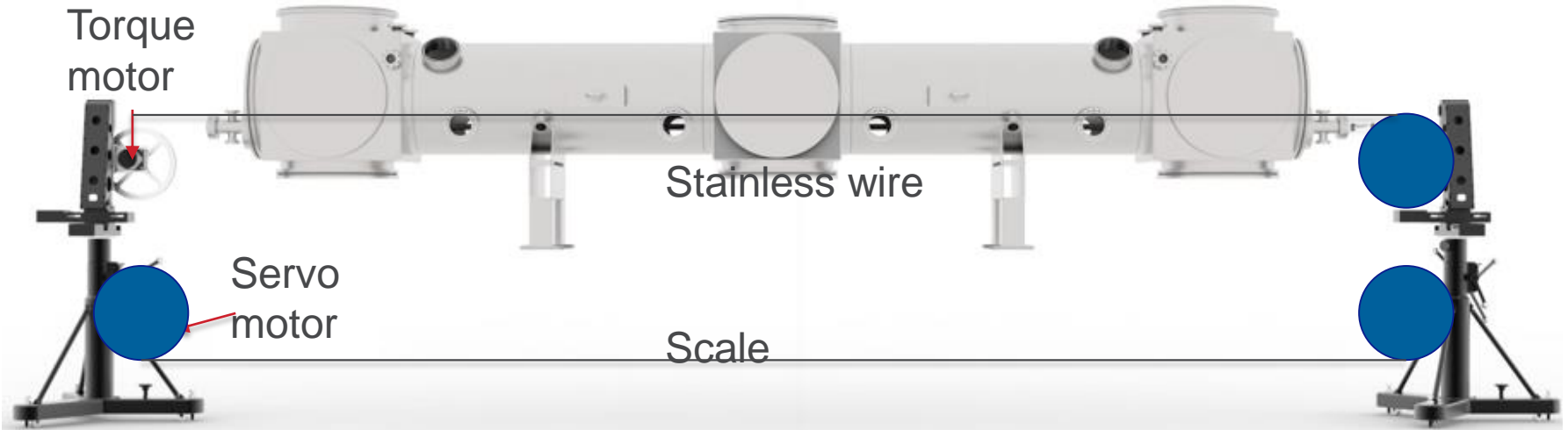


1 V applied to torque motor



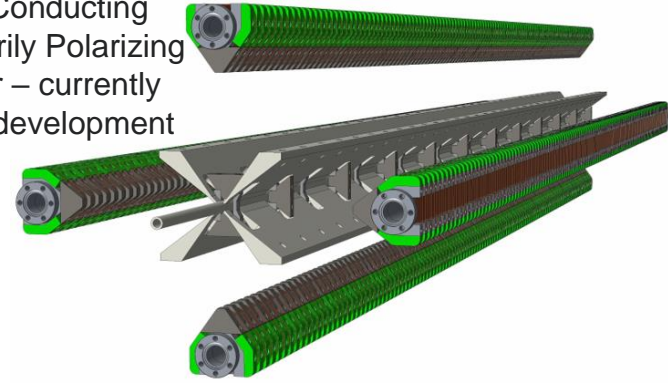
0.85 V applied to torque motor

DRIVE SYSTEM – ALTERNATIVE SETUP



- Move the scale outside of the measurement aperture – space constraint or scale material
- Modification of the guide tube extrusion can be made to accommodate measurement of a permanent magnet undulator with side access to the gap

SuperConducting
Arbitrarily Polarizing
Emitter – currently
under development



CONCLUSION

- A new magnetic measurement system has been developed for the SCU program in preparation for the APS upgrade
- Novel idea for a Hall probe drive system has been developed and initial results look promising
- Wire based measurement system being implemented is similar to the system in MM1
- The system is compact and portable
 - Could be transported to measure IDs outside of the magnetic measurement lab using a different guide tube design



Thank you!